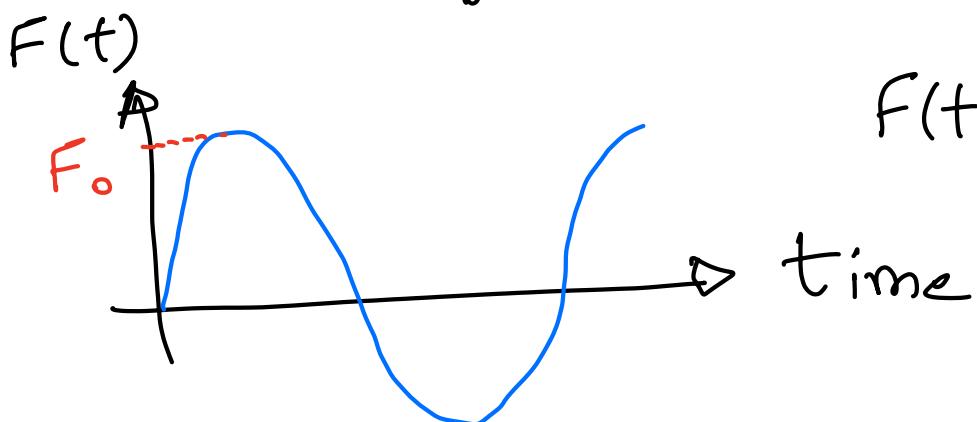
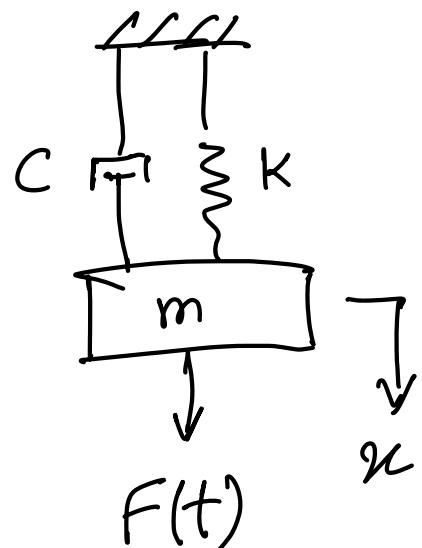
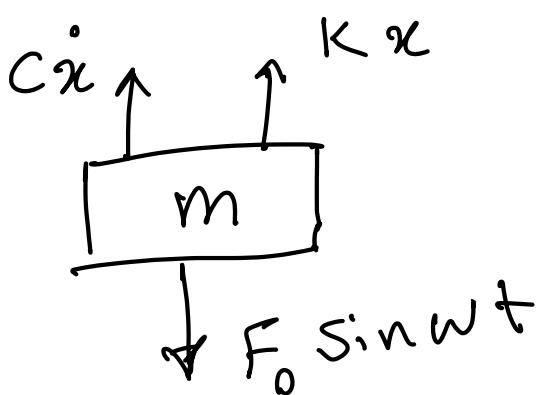


chapter 3

Harmonically Excited vibration

First we consider single-DOF system with viscous damping.

F.B.D



$$F(t) = F_0 \sin \omega t$$

$$m\ddot{x} + c\dot{x} + kx = F_0 \sin \omega t$$

The solution to this equation consists of two parts, the

complementary function, which is the solution of the homogeneous equation, and the particular part.

The complementary is a damped FREE vibration that was discussed in Chapter 2.

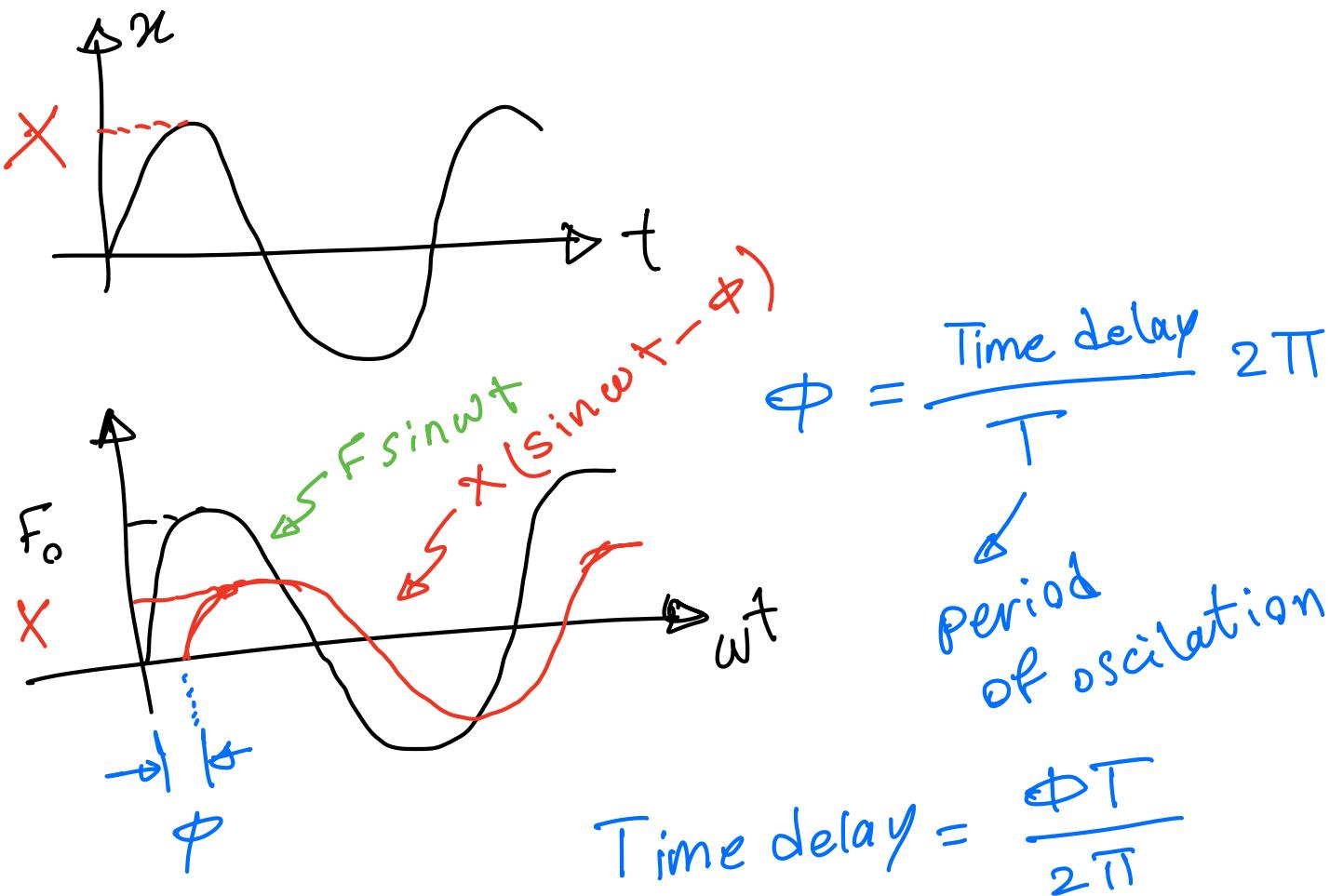
The particular solution is a steady-state oscillation of the same frequency ω as the excitation ($F_0 \sin \omega t$)

The particular solution:

$$x = X \sin(\omega t + \phi)$$

The amplitude
of oscillation

phase of
the displacement
with respect
to the exciting
force.



Excitation force:

$$F_0 (\cos \omega t + i \sin \omega t) = F_0 e^{i \omega t}$$

$$i = \sqrt{-1}$$

$$\begin{aligned}
 x &= X e^{i(\omega t - \phi)} \\
 &= (X e^{-i\phi}) e^{i\omega t} \\
 &= \bar{X} e^{i\omega t}
 \end{aligned}$$

phase shift

$$\bar{X} = X e^{-i\phi}$$

$$m\ddot{x} + cx + Kx = F(t)$$

$$x = \bar{X} e^{i\omega t}$$

$$\dot{x} = \bar{X} i\omega e^{i\omega t}$$

$$\ddot{x} = -\bar{X} \omega^2 e^{i\omega t}$$

$$\boxed{i \cdot i = -1}$$

$$F(t) = F_0 e^{i\omega t}$$

$$-m\bar{X}\omega^2 e^{i\omega t} + c\bar{X} \dot{i}\omega e^{i\omega t}$$

$$+ K\bar{X} e^{i\omega t} = F_0 e^{i\omega t}$$

$$\cancel{\bar{X} e^{i\omega t}} \left[-m\omega^2 + c\omega i + K \right] = F_0 e^{i\omega t}$$

$$(-\omega^2 m + i c \omega + K) \bar{X} = F_0$$

$$\bar{X} = \frac{F_0}{(K - \omega^2 m) + i(c\omega)}$$

$$z = a + ib$$

Real Imag

$$|z| = \sqrt{a^2 + b^2}$$

$$F_0 = F_0 + 0xi$$

$$|\bar{X}|_{\text{magnitude}} = \frac{|F_0|}{|(K - \omega^2 m) + i(c\omega)|}$$

$|F_0| = \sqrt{F_0^2 + 0^2}$
 $= F_0$

$$|\bar{X}| = \frac{F_0}{\sqrt{(K - \omega^2 m)^2 + (c\omega)^2}}$$

natural frequency of undamped oscillation

$$\omega_n = \sqrt{\frac{K}{m}}$$

critical damping

$$c_c = 2m\omega_n$$

Damping factor (ratio) $\xi = \frac{c}{c_c}$ Damping coefficient

$$\frac{c\omega}{K} = \frac{c}{c_c} \frac{c_c \omega}{K} = 2 \xi \frac{\omega}{\omega_n}$$

$z = a + ib$ $\angle z = \tan^{-1} \frac{b}{a}$

, $z_1 - \angle z_1 - \angle z_2$

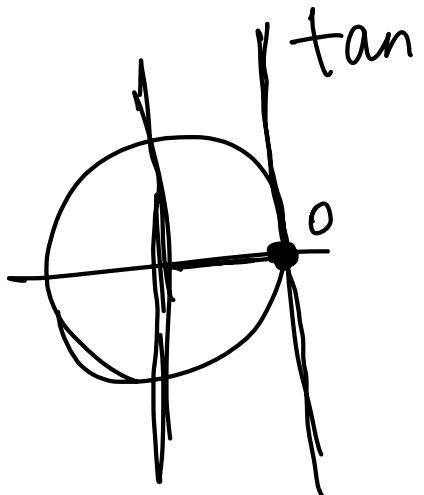
$$\angle \bar{X} = \frac{F_0 + i 0}{(K - \omega^2 m) + i (c\omega)}$$

Real Imag.

$$\phi = \tan^{-1} \frac{0}{F_0} - \tan^{-1} \frac{c\omega}{K - \omega^2 m}$$

$$\phi = 0 - \tan^{-1} \frac{c\omega}{K - \omega^2 m}$$

$$\phi = -\tan^{-1} \frac{c\omega}{K - \omega^2 m}$$



Instead of $c, K, m \rightarrow w_n, f$
 write
 in terms
 of

$$|\bar{X}| = \frac{F_0}{\sqrt{(K - \omega^2 m)^2 + (c\omega)^2}}$$

Divide the numerator and the denominator by K

$$\bar{X} = \frac{F_0/K}{\sqrt{\left(1 - \frac{mw^2}{K}\right)^2 + \left(\frac{cw}{K}\right)^2}}$$

$$\omega_n = \sqrt{\frac{K}{m}} \quad \text{or} \quad \omega_n^2 = \frac{K}{m}$$

$\therefore \omega_n$

$$f = \frac{c}{C_c}$$

~~$c_c = \sqrt{km}$~~

$$\frac{c}{K}$$

~~c^2/m~~

$$= \frac{c^2/m}{4K}$$

$$C_c = 2mw_n$$

$$\omega_n = \sqrt{\frac{K}{m}}$$

$$\omega_n = \sqrt{\frac{K}{C_c/2w_n}}$$

$$f = \frac{c}{C_c}$$

$$\omega_n^2 = \frac{K(2w_n)}{C_c}$$

$$\Rightarrow \omega_n^2 = \frac{K(2w_n)}{c/g}$$

$$\boxed{\frac{C}{K} = \frac{2\zeta}{\omega_n}}$$

$$\bar{x} = \frac{F_0/K}{\sqrt{\left(1 - \frac{m\omega^2}{K}\right)^2 + \left(\frac{C\omega}{K}\right)^2}}$$

$$\frac{x_K}{F_0} = \frac{1}{\sqrt{\left[1 - \left(\frac{\omega}{\omega_n}\right)^2\right]^2 + \left[2\zeta \frac{\omega}{\omega_n}\right]^2}}$$

$$\phi = -\tan^{-1} \frac{C\omega}{K - \omega^2 m}$$

$$\tan \phi = \frac{-C\omega}{K - \omega^2 m}$$

$$\tan \phi = \frac{-C/K \omega}{1 - \omega^2 m / K} \quad \boxed{\frac{C}{K} = 2\zeta / \omega_n}$$

$$\tan \phi = -\frac{2\xi \frac{\omega}{\omega_n}}{1 - \left(\frac{\omega}{\omega_n}\right)^2}$$

$$\frac{m}{K} = \frac{1}{\omega_n^2}$$

$$\frac{\omega}{\omega_n} = ?$$

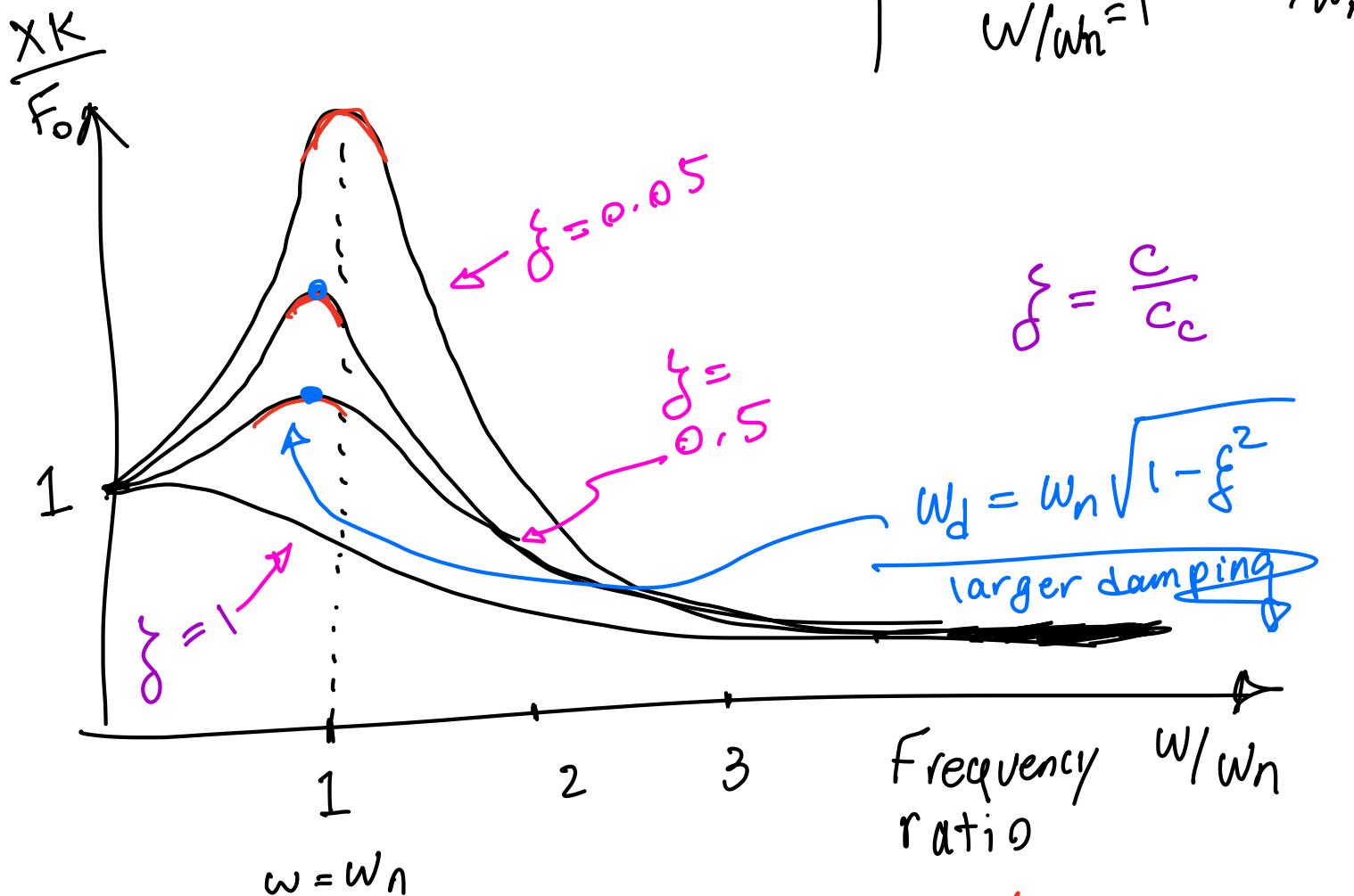
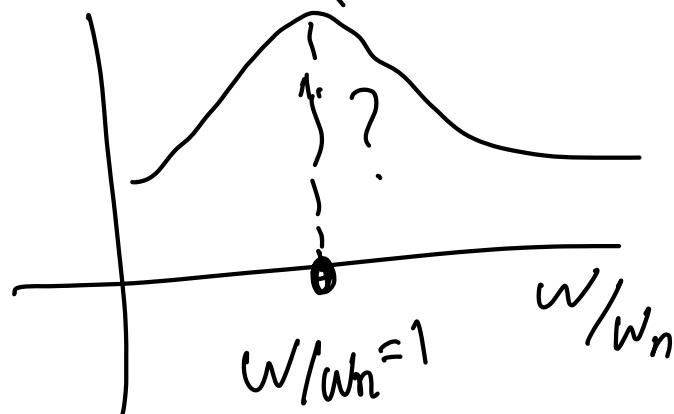
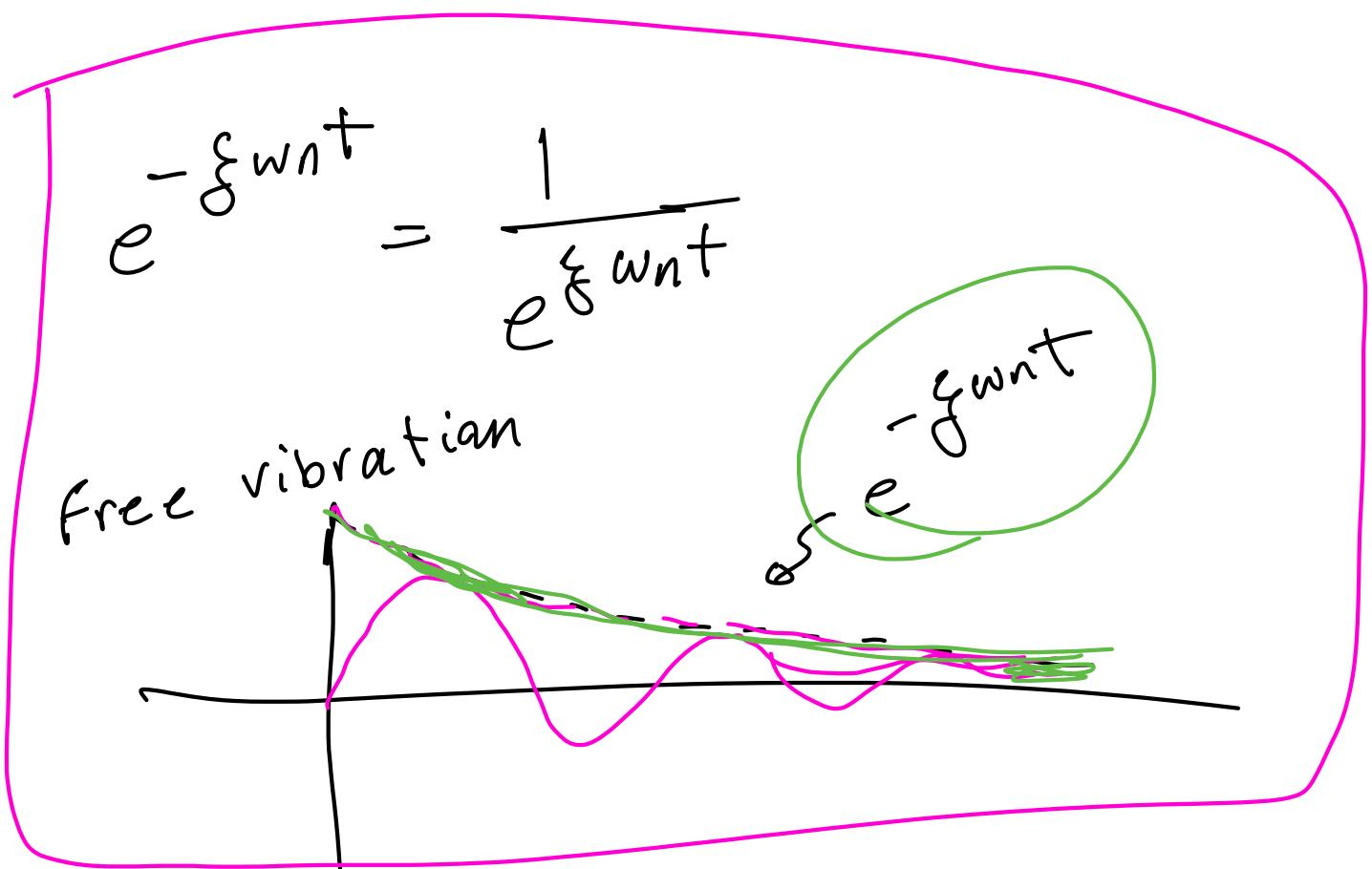


Fig. 3.1-3 Book ↑

$$\ddot{x} + 2\zeta \omega_n \dot{x} + \omega_n^2 x = \frac{F_0}{m} \sin \omega t$$

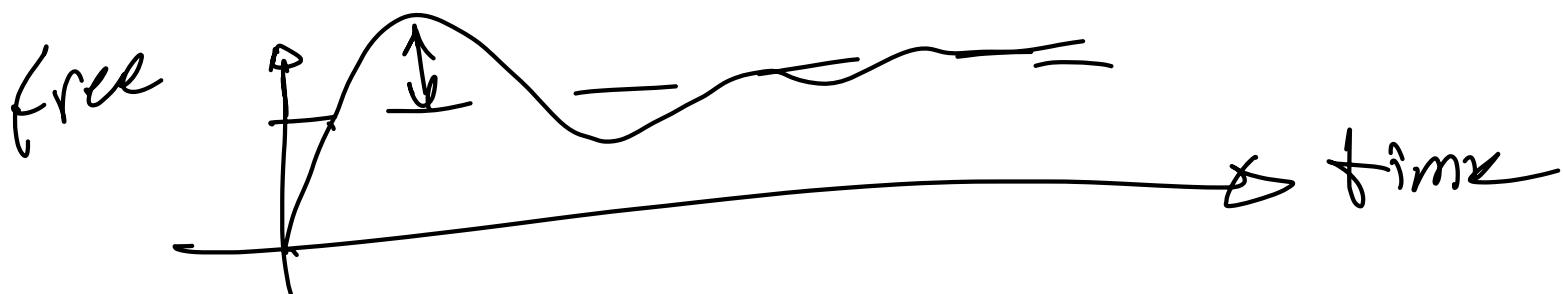
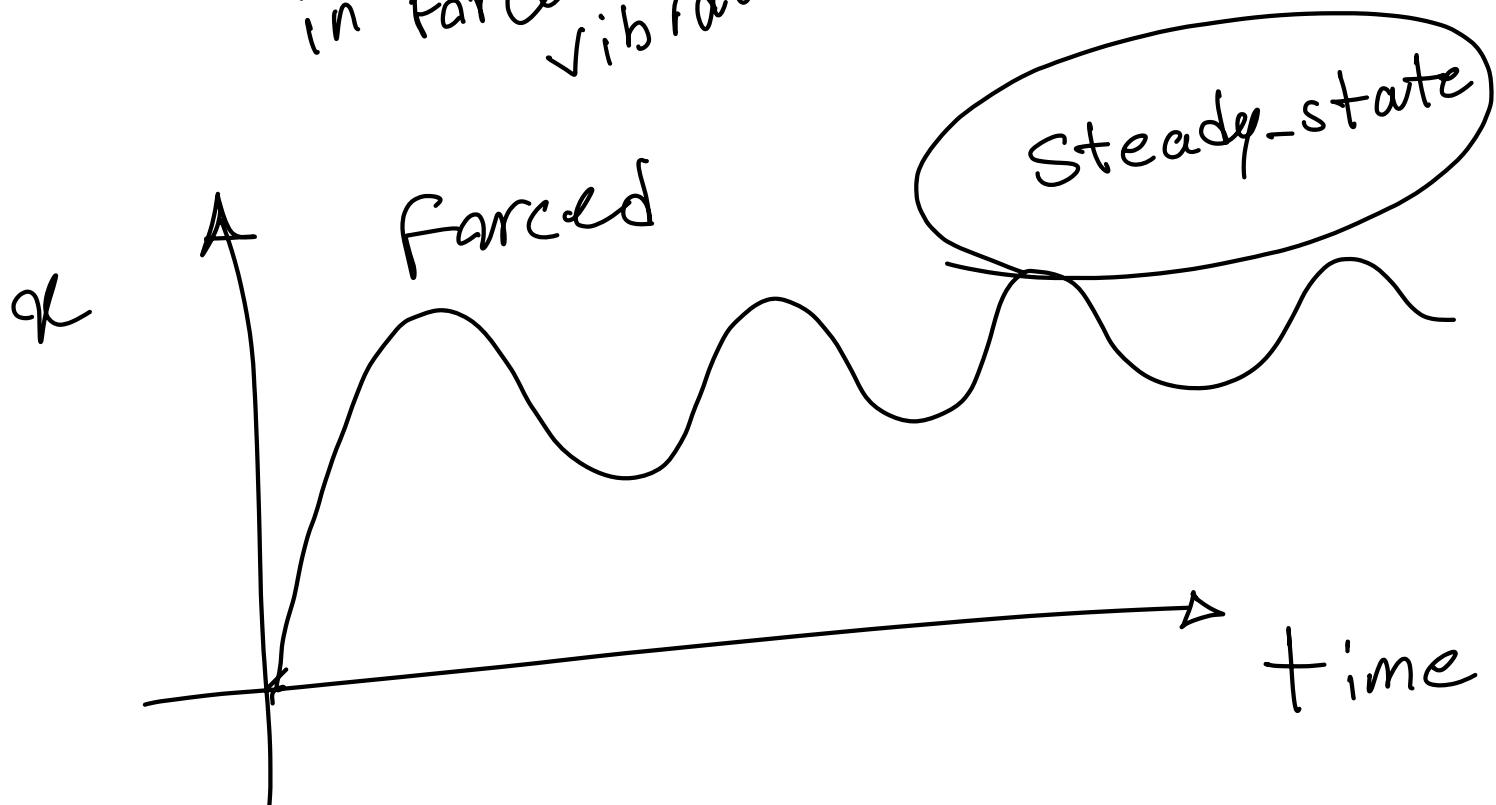
$$x(t) = \frac{F_0}{K} \frac{\sin(\omega t - \phi)}{\sqrt{\left[1 - \frac{\omega}{\omega_n}\right]^2 + \left[2\zeta \frac{\omega}{\omega_n}\right]^2}}$$

$$+ x_1 e^{-\zeta \omega_n t} \sin(\sqrt{1-\zeta^2} \omega_n t + \phi_1)$$



effect
of free
vibration
in forced
vibration

↑
Forced



Next time

Rotating unbalance