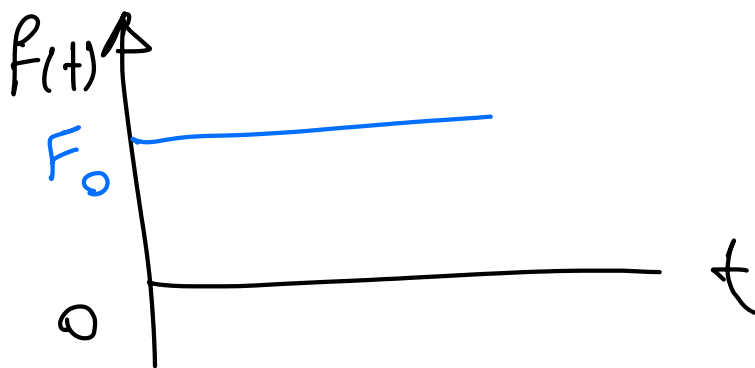


## Example 4.2-1

Determine the response of a single-Dof system to

the step excitation shown below



consider the undamped system

$$h(t) = \frac{1}{m\omega_n} \sin \omega_n t$$

substitute in Eq. (4.2-1)

$$x(t) = \frac{F_0}{m\omega_n} \int_0^t \sin \omega_n (t - \tau) d\tau$$

$$= \frac{F_0}{K} (1 - \cos \omega_n t)$$

For damped system, by repeating the procedure we have

$$h(t) = \frac{e^{-\zeta \omega_n t}}{m \omega_n \sqrt{1-\zeta^2}} \sin \sqrt{1-\zeta^2} \omega_n t$$

For the differential equation

$$\ddot{x} + 2\zeta \omega_n \dot{x} + \omega_n^2 x = \frac{F}{m}$$

solution:

$$x(t) = X e^{-\zeta \omega_n t} \sin(\sqrt{1-\zeta^2} \omega_n t - \phi) + \frac{F_0}{m \omega_n^2}$$

Section 4.3 Laplace transform

formulation. For solving

differential equations.

particularly finding the transfer

function of a system (TF)

$$TF = \frac{\text{output}}{\text{Input}}$$

Example (4.3-1 Book)

The equation of motion of mass-spring-damper with force excitation of  $F(t)$  is

$$m\ddot{x} + c\dot{x} + kx = F(t)$$

Taking the Laplace transform:

from the Appendix:

$$F''(t) \xrightarrow{\text{Laplace Transform}} s^2 F(s) - \underbrace{sf(0) - f'(0)}$$

we usually don't have these initial condition terms.  
 $F'(0) = 0$      $F(0) = 0$

$$f'(t) \xrightarrow{\text{Laplace}} sF(s) - \underline{f(0)} \quad \underline{f(0)} \rightarrow 0$$

Force  $\rightarrow$  Input

$x$   $\rightarrow$  output

Response  
of the mass-spring-damper

$$m\ddot{x} + c\dot{x} + Kx = F(t)$$

Laplace transform:

$$m (s^2 \bar{x}(s) - x(0)s - \dot{x}(0))$$

$$+ c [s \bar{x}(s) - x(0)] + K \bar{x}(s)$$

$$= \bar{F}(s)$$

Solving:

with initial conditions equal to zero we have:

$$m s^2 \bar{x}(s) + c s \bar{x}(s) + k \bar{x}(s) = \bar{F}(s)$$

$$\bar{x}(s) = \frac{\bar{F}(s)}{m s^2 + c s + k}$$

Transfer function  $\left( \frac{\text{output}}{\text{Input}} \right)$

$$\frac{\bar{x}(s)}{\bar{F}(s)} = \frac{1}{m s^2 + c s + k}$$

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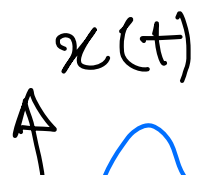
Not using the Book for the rest of the Laplace transform

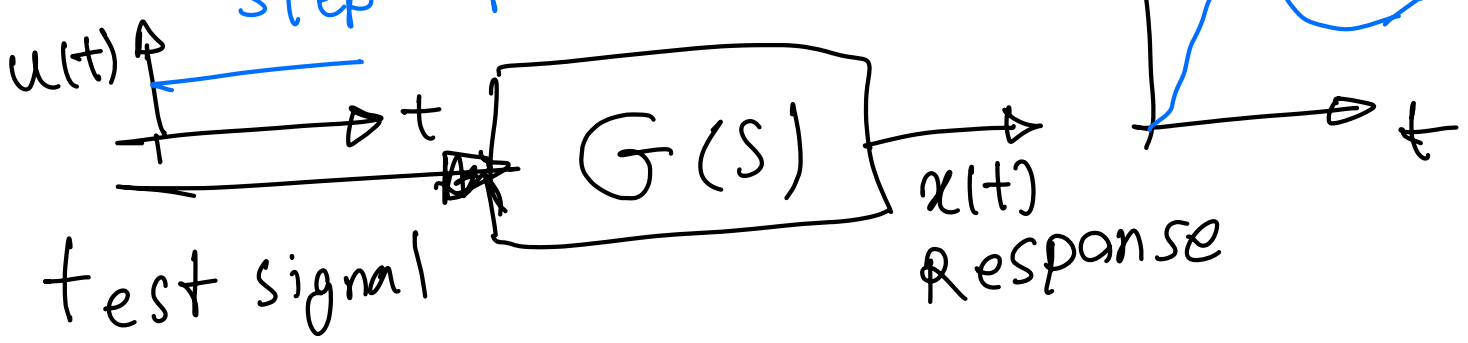
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Time Responses

step input

$x(t)$   
↑





the system response is normally evaluated using standard test input signals

standard test signals:

step input

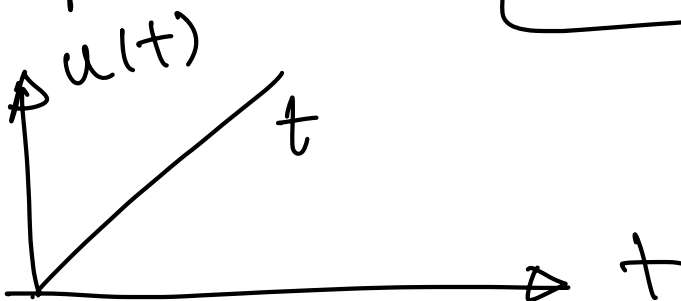


$$h(t) = \begin{cases} 1 & t \geq 0 \\ 0 & t < 0 \end{cases}$$

$$\mathcal{L}[h(t)] = \frac{1}{s}$$

$\uparrow$  Laplace  
 $\mathcal{L}[1] = \frac{1}{s}$

Ramp



$$u(t) = \begin{cases} t & t \geq 0 \\ 0 & t < 0 \end{cases}$$

$$\mathcal{L}[u(t)] = \frac{1}{s^2}$$

$$u(t) = t \xrightarrow{\text{laplace}} \frac{1}{s^2}$$

Mass-spring-damper system

example



$$G(s) = \frac{\text{output}}{\text{Input}} = \frac{X(s)}{U(s)}$$

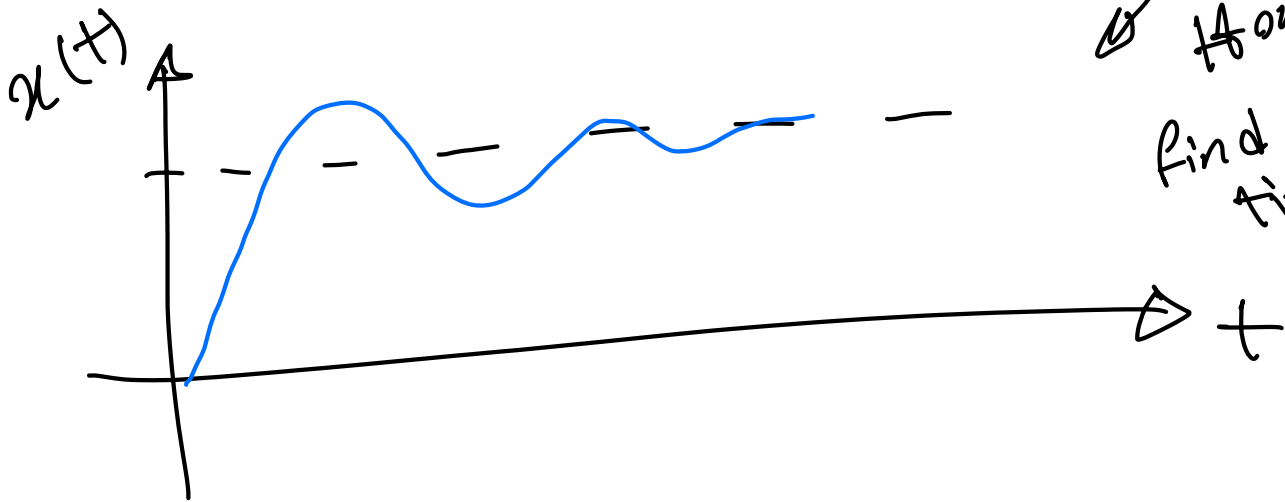
$$M \ddot{x} + B \dot{x} + Kx = u(t)$$

$$G(s) = \frac{1}{Ms^2 + Bs + K}$$

$$M = 800 \text{ kg} \quad K = 400 \text{ N/m}$$

$$B = 300 \text{ Ns/m}$$

$$G(s) = \frac{0.00125}{s^2 + 0.0375s + 0.5}$$



How to  
Find the  
time  
Response

### Example

Find the transfer function  
of  $G(s)$  in time domain.

$$G(s) = \frac{Y(s)}{U(s)} = \frac{6}{s^2 + 5s + 6}$$

Input  $u(t) = 1 \xrightarrow{\text{Laplace}} U(s) = \frac{1}{s}$

$$Y(s) = G(s) U(s)$$



$$Y(s) = \frac{6}{(s^2 + 5s + 6)} \cdot \frac{1}{s}$$

$$Y(s) = \frac{6}{s(s^2 + 5s + 6)}$$

Partial Fraction

$$Y(s) = \frac{6}{s(s+2)(s+3)} = \frac{1}{s} - \frac{3}{s+2} + \frac{2}{s+3}$$

$$y(t) = 1 - 3e^{-2t} + 2e^{-3t}$$

---

Example

$$G(s) = \frac{Y(s)}{U(s)} = \frac{4}{s^2 + 2s + 1}$$

Transfer  
Function

$$u(t) = 1 \xrightarrow{\text{laplace}} U(s) = \frac{1}{s}$$

Input

output

$$Y(s) = \frac{4}{s(s^2 + 2s + 1)} = \frac{4}{s(s+1)^2}$$

partial fraction  $\rightarrow$

$$Y(s) = \frac{A}{s} + \frac{B}{s+1} + \frac{C}{(s+1)^2}$$

$A, B, C = ?$

$$A = 4, \quad B = -4, \quad C = -4$$

$$Y(s) = \frac{4}{s} - \frac{4}{s+1} - \frac{4}{(s+1)^2}$$

$$y(t) = 4 \left( 1 - e^{-t} - te^{-t} \right)$$

using  
Laplace  
table

$$\mathcal{L}^{-1} Y(s) \xrightarrow[\text{table}]{\text{using Laplace}} y(t)$$

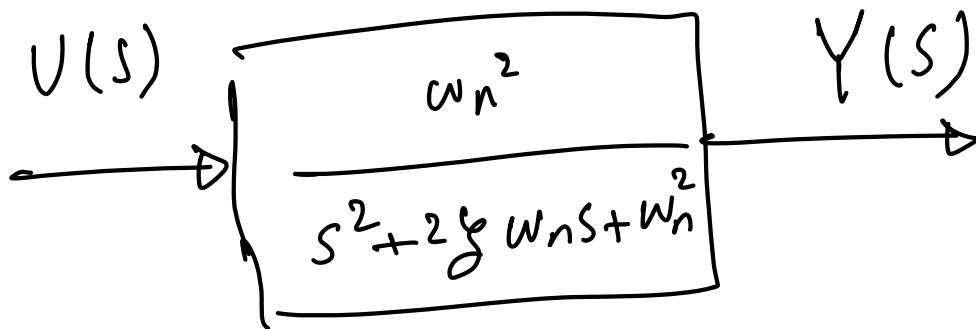
Laplace inverse

Response  
in time domain

## second order transfer function

$$G(s) = \frac{1}{ms^2 + cs + K} = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

↗  
Damping ratio



$$s^2 + 2\zeta\omega_n s + \omega_n^2 = 0$$

$$s^2 + 2\zeta\omega_n s + (\zeta\omega_n)^2 - (\zeta\omega_n)^2 + \omega_n^2 = 0$$

$$\Rightarrow (s + \zeta\omega_n)^2 - (\zeta^2 - 1)\omega_n^2 = 0$$

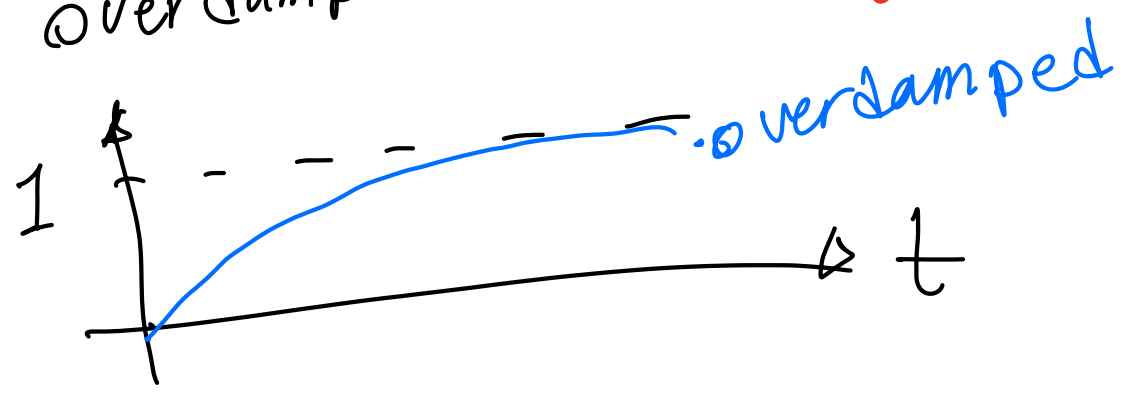
$$\Rightarrow (s + \zeta\omega_n)^2 = (\zeta^2 - 1)\omega_n^2$$

Case 1:  $\zeta > 1$

$$s_{1,2} = -\zeta \omega_n \pm \sqrt{\zeta^2 - 1} \omega_n$$

$$\sqrt{\zeta^2 - 1} > 1$$

overdamped



Case  $\zeta < 1$

$$s_{1,2} = -\zeta \omega_n \pm j \sqrt{1 - \zeta^2} \omega_n$$

under damped response

